

Given: Wet steam throttled adiabatically and aerodynamically

$$P_1 = 800 \text{ psia}$$



$$P_2 = 50 \text{ psia}$$

$$T_2 = 200^\circ\text{F}$$

$$V_1 = V_2, z_1 = z_2$$

Find: The ratio of the exit area to the inlet area $\frac{A_2}{A_1}$.

Solution: The First Law for the throttling process is,

$$\cancel{\dot{Q}} - \cancel{\dot{W}} + \dot{m}_1 \left(h_1 + \frac{V_1^2}{2g_c} + \frac{g}{g_c} z_1 \right) - \dot{m}_2 \left(h_2 + \frac{V_2^2}{2g_c} + \frac{g}{g_c} z_2 \right) = \cancel{\frac{dE_{cv}}{dt}}$$

Continuity: $\dot{m}_1 - \dot{m}_2 = \frac{dm_{cv}}{dt} \rightarrow \dot{m}_1 = \dot{m}_2$

Therefore,

$$h_1 = h_2 \leftarrow \text{throttling is isenthalpic!}$$

How does this relate to the flow areas??

$$\dot{m} = \frac{AV}{v} \rightarrow \therefore \frac{\dot{m}_2}{\dot{m}_1} = \frac{A_2 V_2}{v_2} \cdot \frac{v_1}{A_1 V_1} = \frac{A_2}{A_1} \cdot \frac{v_1}{v_2}$$

Therefore,

$$\frac{A_2}{A_1} = \frac{v_2}{v_1} \cdot \frac{\dot{m}_2}{\dot{m}_1} = \frac{v_2}{v_1}$$

State 2: $\left. \begin{array}{l} P_2 = 5 \text{ psia} \\ T_2 = 200^\circ\text{F} \end{array} \right\} \begin{array}{l} v_2 = 78.15 \text{ ft}^3/\text{lbm} \\ h_2 = 1148.6 \text{ Btu/lbm} \end{array}$ Table C.3a

State 1: $\left. \begin{array}{l} P_1 = 800 \text{ psia} \\ h_1 = 1148.6 \text{ Btu/lbm} \end{array} \right\} \begin{array}{l} h_f = 509.7 \text{ Btu/lbm} \\ h_g = 1199.3 \text{ Btu/lbm} \\ v_f = 0.02087 \text{ ft}^3/\text{lbm} \\ v_{fg} = 0.5691 \text{ ft}^3/\text{lbm} \end{array}$ Table C.2a

State 1 is wet!

$$x_1 = \frac{h_1 - h_f}{h_g - h_f} = \frac{(1148.6 - 509.7) \text{ Btu/lbm}}{(1199.3 - 509.7) \text{ Btu/lbm}} = \underline{0.9265}$$

Therefore,

$$v_1 = (1-x_1)v_f + x_1 v_g$$

$$v_1 = (1-0.9265)(0.02087 \frac{\text{ft}^3}{\text{lbm}}) + (0.9265)(0.5691 \frac{\text{ft}^3}{\text{lbm}}) = \underline{0.5288} \frac{\text{ft}^3}{\text{lbm}}$$

Then,

$$\frac{A_2}{A_1} = \frac{v_2}{v_1} = \frac{78.15 \frac{\text{ft}^3/\text{lbm}}{}}{0.5288 \frac{\text{ft}^3/\text{lbm}}{}} = \underline{147.8} \leftarrow$$

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P[1]=800[psia]
h[1]=h[2]
v[1]=Volume(steam_iapws,P=P[1],h=h[1])
T[1]=Temperature(steam_iapws,P=P[1],h=h[1])
x[1]=Quality(steam_iapws,P=P[1],h=h[1])

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P[2]=5[psia]
T[2]=200[F]
v[2]=Volume(steam_iapws,T=T[2],P=P[2])
h[2]=Enthalpy(steam_iapws,T=T[2],P=P[2])

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Aratio=v[2]/v[1]

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SOLUTION

Unit Settings: Eng F psia mass deg

Aratio = 147.8

No unit problems were detected.

Arrays Table: Main

	h_i [Btu/lbm]	P_i [psia]	T_i [F]	v_i [ft ³ /lbm]	x_i
1	1149	800	518.3	0.5288	0.9263
2	1149	5	200	78.15	

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P[1]=200[psia]
h[1]=h[2]
v[1]=Volume(steam_iapws,P=P[1],h=h[1])
T[1]=Temperature(steam_iapws,P=P[1],h=h[1])
x[1]=Quality(steam_iapws,P=P[1],h=h[1])

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P[2]=5[psia]
T[2]=200[F]
v[2]=Volume(steam_iapws,T=T[2],P=P[2])
h[2]=Enthalpy(steam_iapws,T=T[2],P=P[2])

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Aratio=v[2]/v[1]

SOLUTION

Unit Settings: Eng F psia mass deg

Aratio = 36.3

No unit problems were detected.

Arrays Table: Main

	h_i [Btu/lbm]	P_i [psia]	T_i [F]	v_i [ft ³ /lbm]	x_i
1	1149	200	381.8	2.153	0.9404
2	1149	5	200	78.15	