

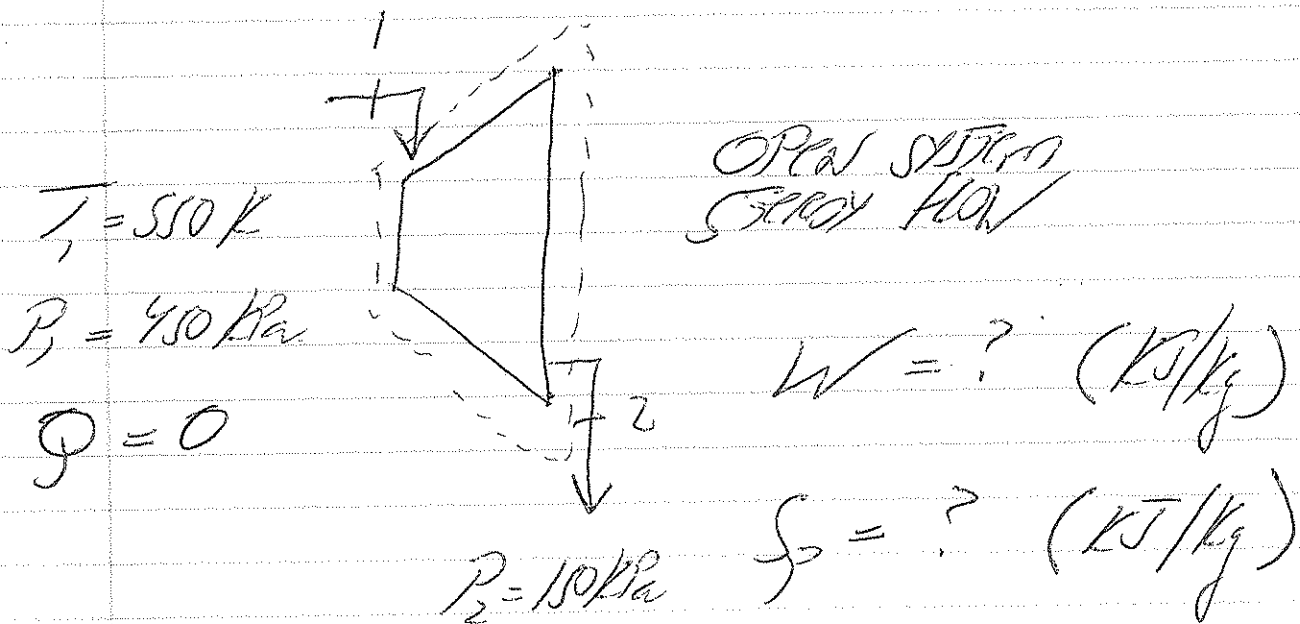
# Example – air turbine w/Air Tables

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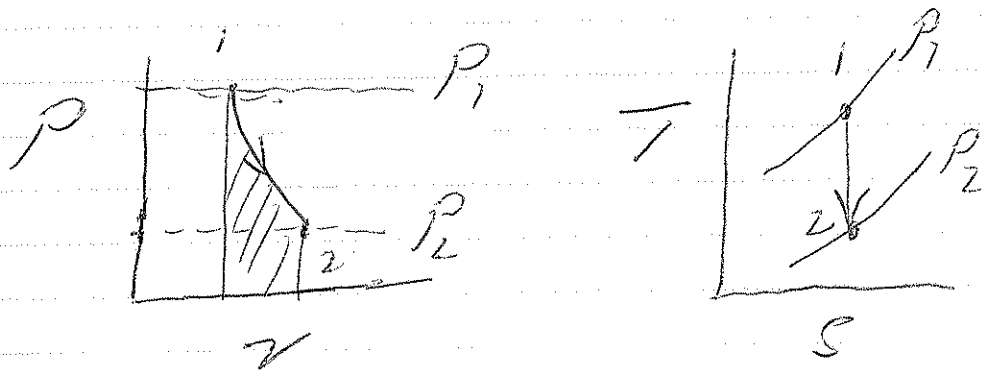
Air expands in an air turbine from a pressure of 450 kPa and a temperature of 550 K to an exhaust pressure of 150 kPa. Assume the process is reversible and adiabatic, with negligible changes in kinetic and potential energy.

- a) Sketch the system & boundary and classify the system.
- b) Show initial state and final state isobars on both P-v and T-s diagrams. Sketch this process on both diagrams. Visualize and forecast sign of work as well as heat transfer.
- c) Use the air tables to find initial and final state conditions.
- d) Write the conservation of energy for this process.
- e) Write the entropy balance for this process.
- f) What is the specific work of air flowing through the turbine?
- g) What is the specific entropy production for this turbine?

# AIR TURBINE PROBLEM



FLUID MODEL: IDEAL GAS, RMC TRAILS



$W$  is +ve

$Q = 0$

PROPERTIES: TABLE C.16 b

$$T_1 = 550 \text{ K} \quad h_1 = 555 \text{ kJ/kg} \quad P_{r1} = 11.86$$

$$P_{r2} = P_{r1} \left( \frac{P_2}{P_1} \right) = 11.86 \left( \frac{150}{450} \right) = 3.95$$

$$T_2 \approx 405 \text{ K} \quad h_2 \approx 405 \text{ kJ/kg}$$

ON SPECIAL ENERGY BASIS

1st LAW

$$\dot{q} - W + (h_i + \frac{V_i^2}{2g_c} + g \dot{z}_i / g_c) - (h_e + \frac{V_e^2}{2g_c} + g \dot{z}_e / g_c) = \frac{dE_{cv}}{dt}$$

so  $W = h_i - h_e = 555 - 405 = 150 \text{ kJ/kg}$

CONTINuity  
ENTROPY

$$s_i = s_e$$

$$\int \frac{\dot{q}}{T_k} + \dot{E}_{f_i} - \dot{E}_{f_e} + \dot{S}_p = 0$$

so  $\dot{S}_p = 0$  WHICH IS SHOWN FOR A REVERSIBLE PROCESS.

ON SPECIAL ENTROPY BASIS

"AIR TURBINE PROBLEM"

"Initial State"

```
f$='air'
T[1] = 550[K]
P[1] = 450[kPa]
s[1] = Entropy(f$, T=T[1], P=P[1])
h[1] = Enthalpy(f$, T=T[1])
```

```
hs[1] = h[1]
ss[1] = s[1]
Ts[1] = T[1]
```

"Final State (Isentropic Process)"

```
P[2] = 140[kPa]
ss[2] = ss[1]
Ts[2] = Temperature(f$, P=P[2], s=ss[2])
hs[2] = Enthalpy(f$, T=Ts[2])
```

"Isentropic Process Analysis"

```
q = 0[kJ/kg]
q - ws + hs[1] - hs[2] = 0 "Energy Conservation"
```

```
Tb = 300[K]
Sps = -q/Tb + (ss[2]-ss[1]) "Entropy Balance"
```

"Analysis with nt = 80%"

```
nt = .80[dim]
nt = wa/ws
q-wa + h[1] - h[2] = 0
T[2] = Temperature(f$, h=h[2])
s[2] = Entropy(f$, T=T[2], P=P[2])
Sp = -q/Tb + (s[2] - s[1])
```

SOLUTION

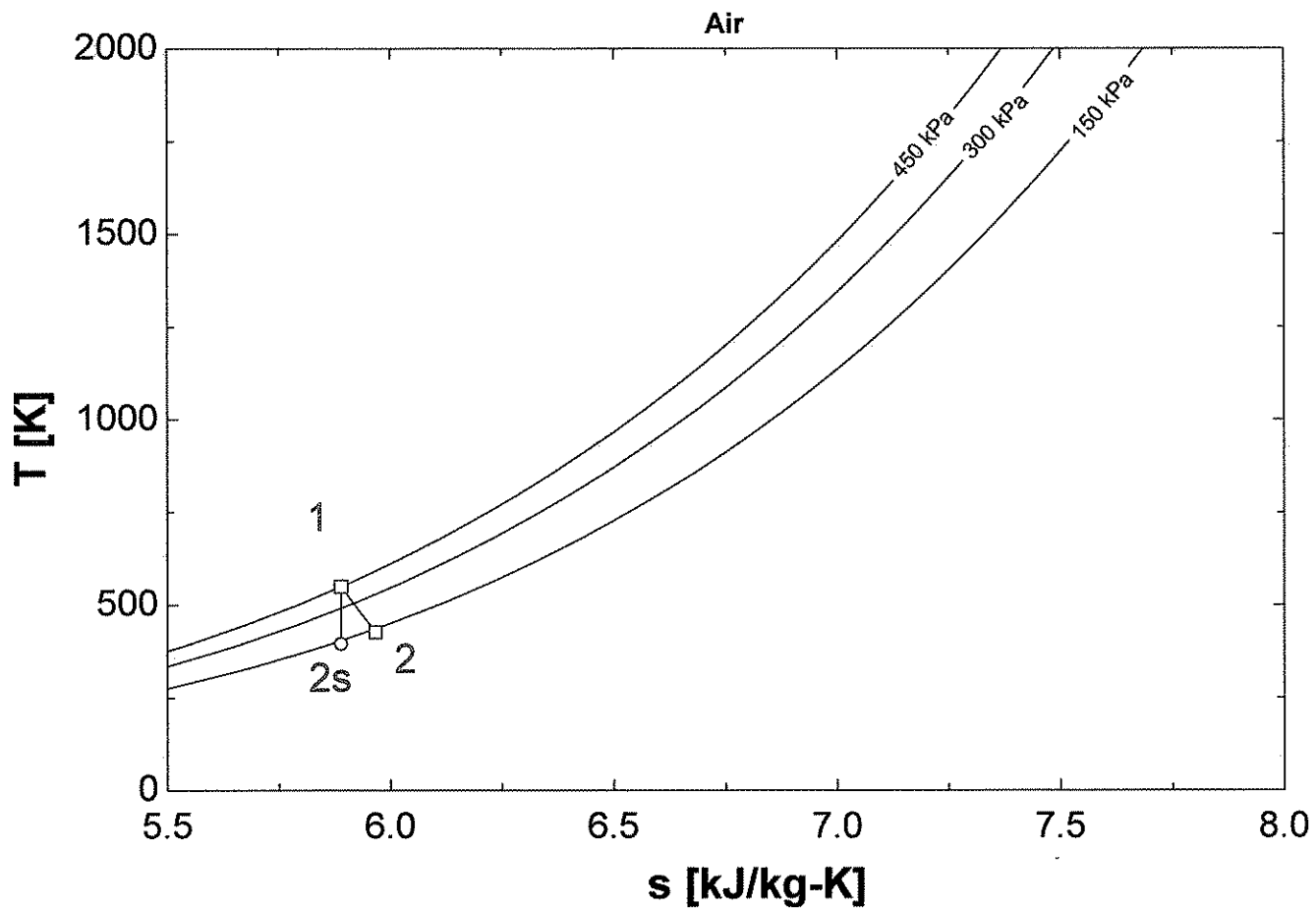
Unit Settings: SI K kPa kJ mass deg

```
f$ = 'air'           nt = 0.8 [dim]           q = 0 [kJ/kg]
Sp = 0.0764 [kJ/kg-K] Sps = 0 [kJ/kg-K]       Tb = 300 [K]
wa = 126 [kJ/kg]     ws = 157.5 [kJ/kg]
```

No unit problems were detected.

Arrays Table: Main

	P <sub>i</sub>	s <sub>i</sub>	T <sub>i</sub>	h <sub>i</sub>	Ts <sub>i</sub>	hs <sub>i</sub>	ss <sub>i</sub>
	[kPa]	[kJ/kg-K]	[K]	[kJ/kg]	[K]	[kJ/kg]	[kJ/kg-K]
1	450	5.889	550	555.1	550	555.1	5.889
2	140	5.966	427.4	429	396.3	397.5	5.889



REFLECTION:

- 1) Note slight difference between hand calculations with Air Tables and EES ideal gas model.
- 2) Create Ts Property Plot, specifying initial pressure, intermediate pressure, and final pressure.
- 3) Overlay Ts versus ss and T versus s to show ideal expansion and real expansion processes.
- 4) Click on each axis to modify min and max values, making it easier to distinguish between ideal and real processes.